

Isogeometric analysis for large deformation contact problems

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Introduction

Isotropic Hyperelastic materials

- Strain energy function to describe nonlinear mechanical behaviour
- Same response of the material in all directions
- Mooney Rivlin material as a special case of the Ogden material

Discrete contact formulation

- NTS-discretization as five-node contact element
- Active set strategy to fulfill the Kuhn-Tucker-Karush condition
- Lagrange multipliers to add contact contributions to the weak form

Isogeometric analysis

- NURBS based shape functions
- KTS-method to enforce contact
- Definition of an exact torus geometry

Constitutive law

Mooney-Rivlin material [1]

- Strain energy function

$$\Psi(I_1, I_2, J) = c_1(I_1 - 3) + c_2(I_2 - 3) - b \ln(J) + c(J - 1)^2$$

- Growth conditions

$$\Psi(J \rightarrow +\infty) \rightarrow +\infty \quad \text{and} \quad \Psi(J \rightarrow 0^+) \rightarrow +\infty$$

- Second Piola-Kirchhoff stress tensor and elasticity tensor

$$\mathbf{S} = 2 \frac{\partial \Psi}{\partial \mathbf{C}} \quad \mathbb{C} = 4 \frac{\partial^2 \Psi}{\partial \mathbf{C}^2}$$

Contact formulation

Constraints

- Non-penetration condition

$$g_N = (\mathbf{x}^S - \bar{\mathbf{x}}^M) \cdot \bar{\mathbf{n}}^M \geq 0$$

- Kuhn-Tucker-Karush condition

$$g_N \geq 0 \quad p_N \leq 0 \quad g_N p_N = 0$$

- Lagrange multiplier method

$$\Pi_c = \int_{\Gamma_c^{akt}} p_N g_N \, d\Gamma$$

NTS-discretization

- Interpolation of Master-surface

$$\mathbf{x}_h^M(\xi_1, \xi_2, t) = \sum_{i=1}^4 \hat{N}_i(\xi_1, \xi_2) \mathbf{x}_i^M(t)$$

- Five-node-contact element

$$\mathbf{x}_{(s)} = [\mathbf{x}^S \mathbf{x}_1^M \mathbf{x}_2^M \mathbf{x}_3^M \mathbf{x}_4^M]_{(s)}^T$$

- Weak form of contact contribution

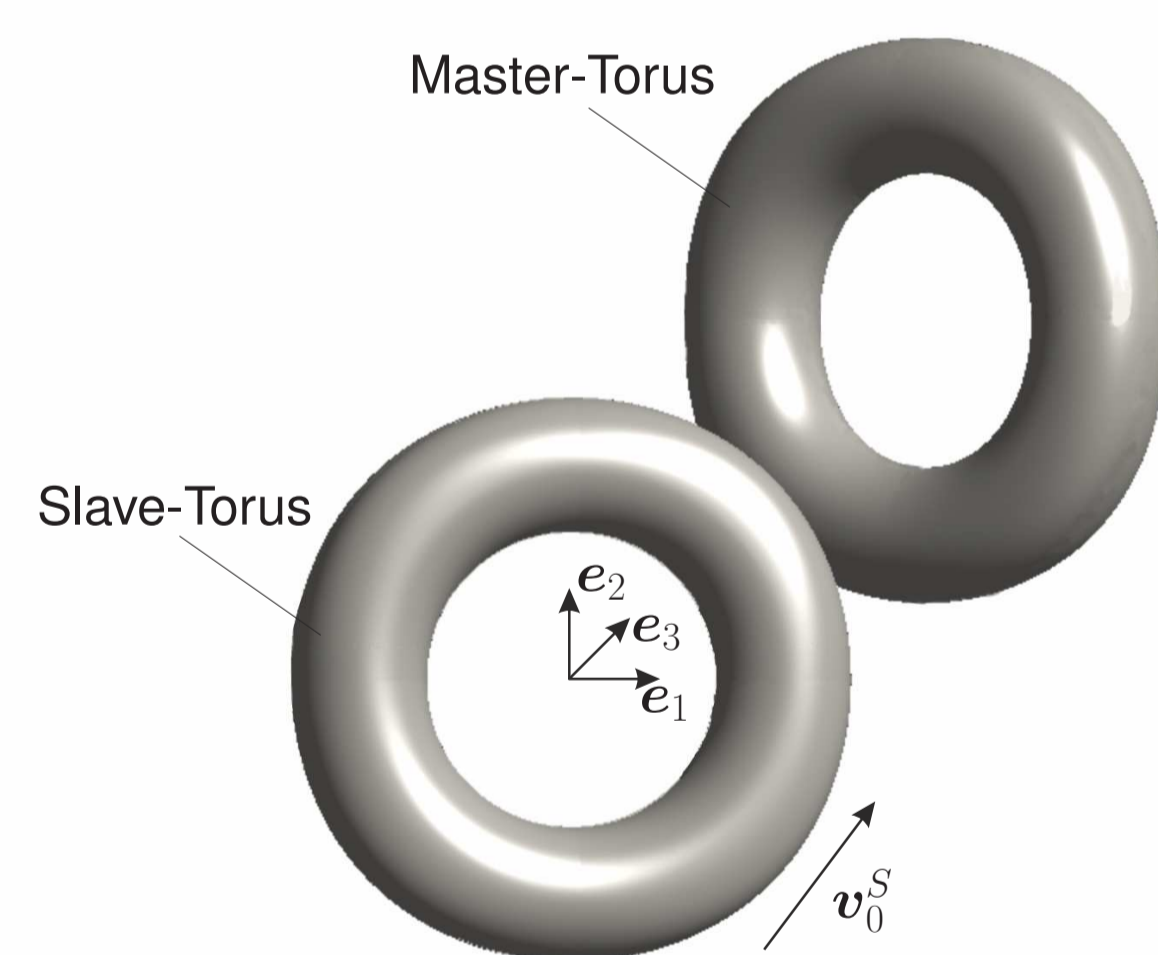
$$\mathbf{C}_c = \sum_{s=1}^{n_c} p_{N(s)} \delta g_{N(s)} \mathbf{A}_{(s)} = \sum_{s=1}^{n_c} \boldsymbol{\eta}_{(s)}^T \mathbf{C}_{(s)}$$

- Discrete contact constraints

$$\boldsymbol{\Phi} = [g_{N(1)}, \dots, g_{N(s)}, \dots, g_{N(n_c)}]^T = \mathbf{0}$$

- Weak form

$$G_h(\mathbf{x}, \boldsymbol{\eta}) = \boldsymbol{\eta}^T [\mathbf{M}\ddot{\mathbf{x}} + \mathbf{R}(\mathbf{x}) + \mathbf{C}(\mathbf{x})] = 0$$



Discretization in time

Mid-point type rule

$$\begin{aligned} \mathbf{x}_{n+1} - \mathbf{x}_n &= \Delta t \mathbf{v}_{n+\frac{1}{2}} \\ \mathbf{M}(\mathbf{v}_{n+1} - \mathbf{v}_n) &= -\Delta t [\mathbf{R}(\mathbf{x}_{n+1}, \mathbf{x}_n) + \mathbf{C}(\mathbf{x}_{n+1}, \mathbf{x}_n)] \\ \mathbf{0} &= \boldsymbol{\Phi}(\mathbf{x}_{n+1}) \end{aligned}$$

Isogeometric analysis

NURBS [2]

- Shape functions

$$R_{i,j,k}^{p,q,r} = \frac{N_{i,p}(\xi) M_{j,q}(\eta) L_{k,r}(\zeta) w_{i,j,k}}{\sum_{i=1}^n N_{i,p}(\xi) \sum_{j=1}^m N_{j,q}(\eta) \sum_{k=1}^l N_{k,r}(\zeta) w_{i,j,k}}$$

- B-Splines basis functions are recursively defined as

$$N_{i,p}(\xi) = \frac{\xi - \xi_i}{\xi_{i+p} - \xi_i} N_{i,p-1}(\xi) + \frac{\xi_{i+p+1} - \xi}{\xi_{i+p+1} - \xi_{i+1}} N_{i+1,p-1}(\xi)$$

- Beginning with

$$N_{i,0}(\xi) = \begin{cases} 1 & \text{if } \xi_i \leq \xi \leq \xi_{i+1}, \\ 0 & \text{otherwise} \end{cases}$$

- Knot vectors

$$\Xi = [\xi_1, \dots, \xi_{n+p+1}] \quad \mathcal{H} = [\eta_1, \dots, \eta_{m+q+1}] \quad \mathcal{Z} = [\zeta_1, \dots, \zeta_{l+r+1}]$$

- Control points

$$\tilde{\mathbf{B}} = [\tilde{\mathbf{B}}_{1,1,1}, \dots, \tilde{\mathbf{B}}_{i,j,k}, \dots, \tilde{\mathbf{B}}_{n,m,l}]^T$$

- NURBS Solid

$$\mathbf{B}(\xi, \eta, \zeta) = \sum_{i=1}^n \sum_{j=1}^m \sum_{k=1}^l R_{i,j,k}^{p,q,r}(\xi, \eta, \zeta) \tilde{\mathbf{B}}_{i,j,k}$$

Numerical results

Impact of two hollow tori

- Material parameter

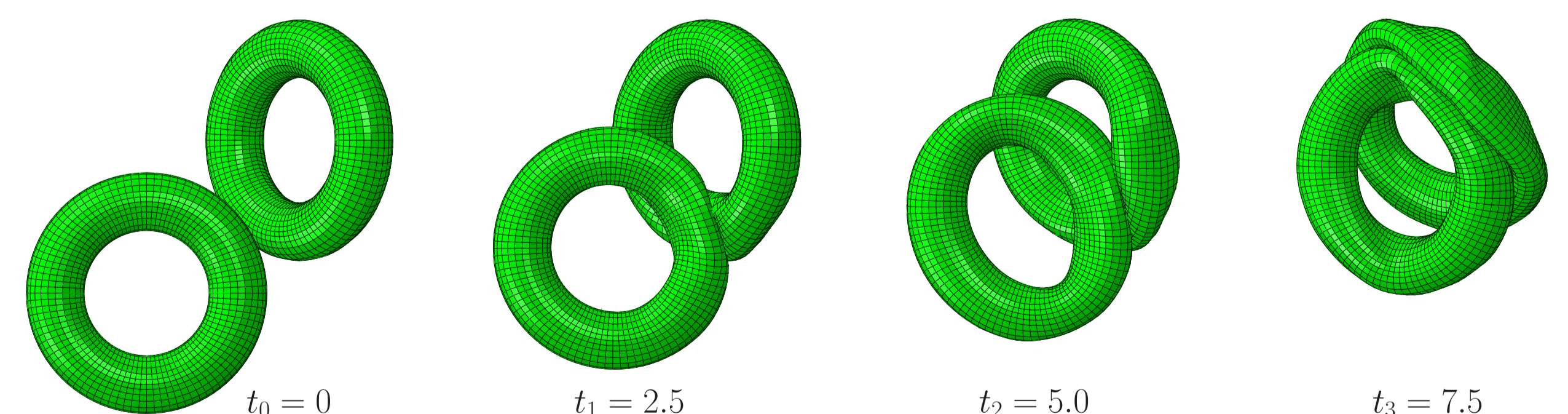
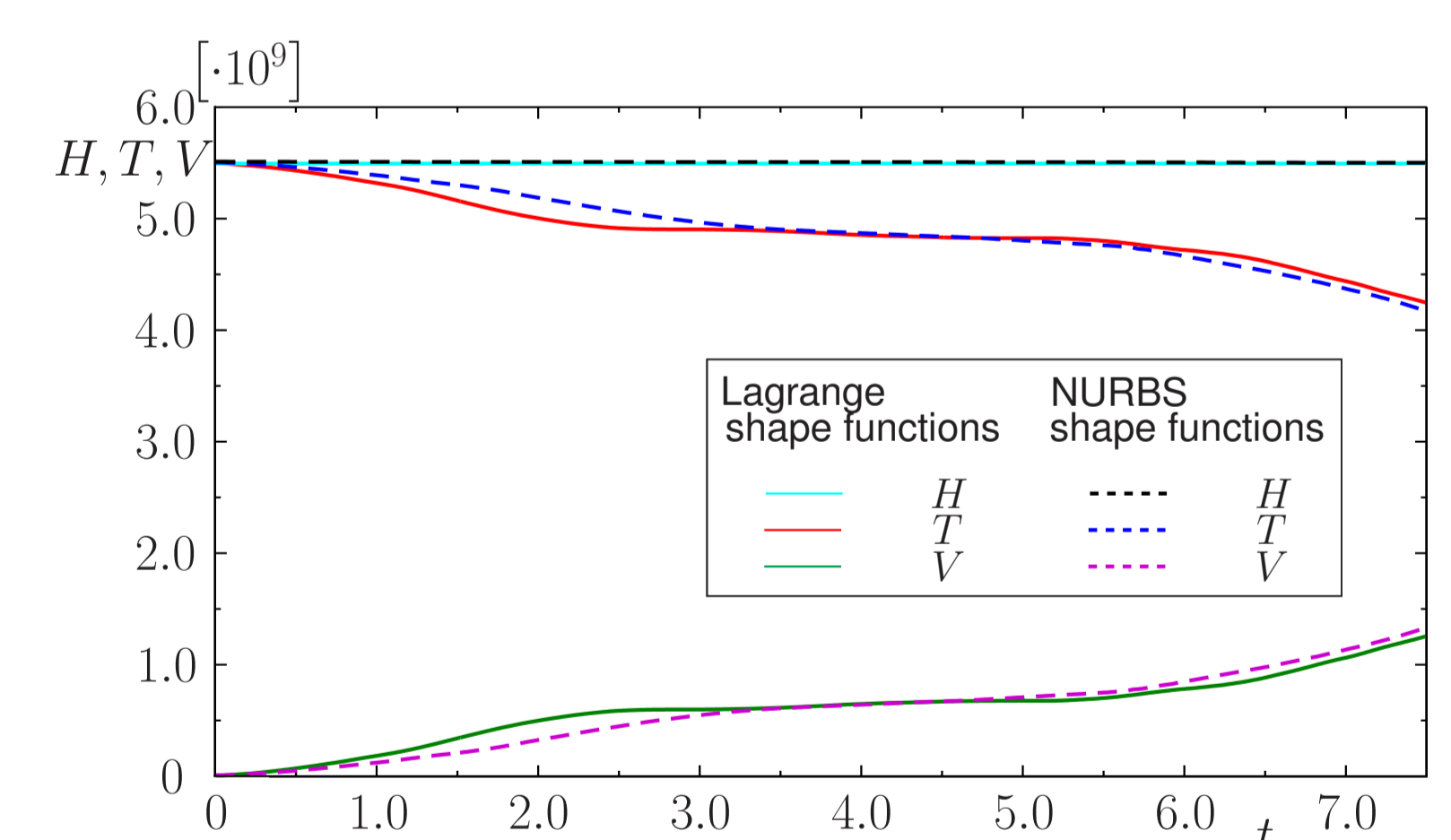
$$E = 1.0507 \cdot 10^6$$

$$\nu = 0.4562$$

$$\rho_0 = 60$$

- Initial velocity

$$\mathbf{v}_0^S = [20 \ 15 \ 0]^T$$



References

- G. A. Holzapfel, Nonlinear Solid Mechanics. Wiley, Chichester, 2000
- T.J.R. Hughes, J.A. Cottrell, Y. Bazilevs, Isogeometric analysis: CAD, finite elements, NURBS, exact geometry and mesh refinement. *Comput. Methods Appl. Mech. Engrg.*, 194:4135-4195, 2005